





Modeling and Analysis of SNF Transportation, a Summary

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This Work Relative to other DOE NE R&D Efforts

Conditions and Structural Capacities of Used Nuclear Fuel (UNF)

High Burn up Demo:

- Temperatures are low.
- Good for cladding.
- CIRFT (Cladding Fatigue) Testing:
 - Fuel contributes.
- Cladding irradiated fatigue strength is better than expected.

Sister Rod Mechanical Testing:

- **Cladding properties**
- Composite rod properties.

Normal Conditions of Transport (NCT) Shock and Vibration Loads

Multimodal Transportation Test (MMTT) Campaign:

- Recorded NCT shock and vibration environment
- Measured cladding strains
- Validate structural & dynamic models
- Use validated analytical methods to predict response of other systems



All DOE-NE R&D is laying the foundation for this conclusion:

Irradiated fuel rod conditions and structural capacity >> NCT transportation loads, therefore it is safe to ship UNF and to expect it to arrive with no additional damage.

Key DOE NE Reports on MMTT

- McConnell PE, SB Ross, CA Grey, WL Uncapher, M Arviso, R Garmendia, IF Perez, A Palacio, G Calleja, D Garrido, AR Casas, LG Garcia, W Chilton, DJ Ammerman, J Walz, S Gershon, SJ Saltzstein, K Sorenson, NA Klymyshyn, BD Hanson, R Pena, and R Walker. 2018. *Rail-Cask Tests: Normal-Conditions-of- Transport Tests of Surrogate PWR Fuel Assemblies in an ENSA ENUN 32P Cask*. SFWD-SFWST-2017-000004, Sandia National Laboratories, Albuquerque, New Mexico.
- Kalinina, EA, C Wright, N Gordon, SJ Saltzstein, L Lujan, KM Norman.
 2018. Data Analysis of ENSA/DOE Rail Tests. 2018. SFWD-SFWST-2018-000494, Sandia National Laboratories, Albuquerque, New Mexico.
- Klymyshyn N.A., P. Ivanusa, K. Kadooka, C.J. Spitz, P.J. Jensen, S.B. Ross, B.D. Hanson, D. Garcia, J. Smith, S. Lewis. 2018. *Modeling and Analysis of the ENSA/DOE Multimodal Transportation Campaign. 2018.* PNNL-28088. Richland, WA: Pacific Northwest National Laboratory.
 - This presentation is a summary of this report. See this report for additional information.

Presentation Objectives

- Show a selection of the MMTT test results.
 - Select key data for discussion (Westbound Rail).
- Provide context and perspective to the test results.
 - Fuel cladding stress, strain, load, deformation energy are all low.
- Describe the progress of structural-dynamic modeling:
 - Validated structural-dynamic models to predict loads on other transportation system configurations.
 - Fatigue analyses.
 - Next steps.

Strain Energy Perspective: UNF Rod Shock and Vibration Energy Comparisons

Structural-dynamic models predict that the strain energy implied by the strain values recorded on the fuel cladding is so low that it is comparable to the kinetic energy in one raindrop.

| Moving Object | Specific Example | Kinetic Energy (mJ) | |
|--------------------|---|---------------------|--------|
| Bullet | AR-15 | 1,854,000.0 | |
| (Muzzle Energy) | 9 mm Handgun | 467,000.0 | |
| Golf Ball | PGA Tour, Male (168 mph) | 129,000.0 | |
| (Off the Tee) | Amateur, Bogey Golfer (131 mph) | 77,000.0 | |
| Bird Flying | Robin (25 mph) European Swallow (19 mph) | 4,400.0 1,200.0 | |
| Ping Pong Ball | World Record (70 mph) | 1,300.0 | |
| (Table Tennis) | Average (25 mph) | 168.0 | |
| Single Raindrop | Heavy Thundershower (130 mg, 20 mph) Moderate Rain (37 mg, 17 mph) Light Drizzle (8 mg, 14 mph) | 5.2 1.0 0.1 | ٦ |
| Fuel Rod Vibration | Single Rod Model Estimate (50 mph P&B) | 1.3 | \int |
| (Strain Energy) | Gravity | 0.7 | |
| Flying Insect | Wasp (15 mph) Housefly (4 mph) | 2.2 < 0.1 | |

Select MMTT Results: Westbound Rail Summary



- Rail transportation is the most important mode in the US because that is how the bulk of UNF is expected to be moved.
- Cladding strains are the key measurement because they tell us about the deformation of the rods. Deformation relates to stress, strain, and strain energy.
- Cladding strains are so small that it is convenient to discuss them in units of microstrain (uE).
 - 1 uE = 0.000001 in/in, or 0.0001%

Westbound Rail Transportation Peaks



- The peak strain of 46 uE is attributed to PLN. 37 uE is the peak when PLN is reduced by a PNNL algorithm.
- Platform is the railcar deck center. The corner railcar deck accelerometers are excluded due to excess vibrations.

Strain Gage and Assembly Color Key

Three instrumented fuel assembly locations.



- The ENUN 32 P cask used in the MMTT holds 32 fuel assemblies.
- Three instrumented fuel assemblies were used in the test.
- 29 simulated assemblies filled the remaining baskets.

Three common strain gage locations.



 There is one fuel rod location on all 3 fuel assemblies

Example Westbound Rail Strain Gage Data (1 hour)

- This 1-hr section of data includes a road crossing that causes the peak recorded cask vertical acceleration. Notice that there is no clear indication of this event in the strain data.
- Strain peaks above 10 uE can be traced to track features, like road crossings.



Example Westbound Rail Data between Events (1 minute)

- This 1-minute section of data shows strain cycles with an amplitude less than 1 uE.
- The fatigue discussion will note 50 million strain cycles like these. Note that they are too small to cause any practical fatigue damage.



Strain Gage Frequency Spectra



Rail Transportation Peak Cladding Strains

Captive Track Test Peak Strains Compared to Open Rail

| | | Maximum Strain (uE) | |
|-------------------------------------|--------------|---------------------|-----------------|
| Test | SNL Assembly | Spanish Assembly | Korean Assembly |
| Coupling Impact | 96.4 | 74.1 | 39.0 |
| Diamond Crossing | 32.8 | 15.5 | 6.7 |
| Dynamic Curving | 36.7 | 22.3 | 8.3 |
| Hunting | 37.2 | 18.6 | 10.4 |
| PCD | 32.3 | 13.6 | 5.5 |
| Single Bump | 37.0 | 20.2 | 9.1 |
| Twist and Roll, Pitch and Bounce | 29.3 | 24.8 | 13.1 |
| All Open Westbound Rail | 36.5 | 27.9 | 36.5 |

- PLN is reduced in Open Westbound Rail category
- Captive track tests bound the open rail response.
- Peak strain energy in each rod is small: around 1 mJ
- Amplification of 2-3 orders of magnitude would be needed to challenge fuel cladding strength.

Structural-Dynamic Modeling Progress

- The test data was used to validate dynamic and structural models:
 - NUCARS rail dynamics models predict conveyance system motion.
 - LS-DYNA explicit dynamic finite element models of a full fuel assembly and a single fuel rod.
- A minimal model architecture was developed to predict cladding strains. This model agrees well with test data and is economical to solve.
 - The minimal model can always be expanded as necessary, but the loads are close to zero.
- Additional analyses were made to investigate conveyance system components and their effect on the test.
 - Simulated fuel assemblies
 - Modal analysis of cask and cradle system.

Minimal Model Architecture – 2 Parts



Single Rod Model Validation

50 mph Pitch and Bounce Test Validation Case

| Case | Peak Strain at Gage (uE) | Maximum Strain Cycle Amplitude (uE) | Number of Strain Cycles over 10 uE | <u>2-Part Baseline Model</u>: All information comes from models. Pure model estimate of strains. |
|-----------------------------|-----------------------------------|--|---|---|
| Baseline 2-Part Model | 29.9 | 29 | 167 | <u>1-Part Model (A15Zv)</u>: Cask motion comes from recorded test data. This case represents the ability of the model to |
| 1-Part Model (A15Zv) | 11.9 | 12 | 6 | calculate strains when cask motion prediction is perfect. |
| Actual SG4 | 11.3 | 11 | 2 | <u>Actual SG17</u>: Strain gage data. (SNL) |
| Actual SG17 | 7.3 | 7 | 0 | <u>Actual SG28</u>: Strain gage data. (Korean^O) |
| Actual SG28 ^O | 5.3 | 6 | 0 | |

- The baseline 2-part model provides conservative results compared to test data.
- The 1-part model provides better agreement, but it uses test data. There may be room to improve the 2-part model further.

Modeling Study: Simulated Assemblies



2. Frequency sweep analyses

Simulated Assembly Studies:

1. **Modal analysis** shows response is similar to real fuel assemblies.

2. Frequency sweep analyses

demonstrate response frequencies similar to real fuel assemblies in horizontal position. (Damping is low.)

3. Basket shock pulse analyses show dynamics expected of real fuel.

Conclusion: Simulated assemblies do not have a strong effect on test results.

Fatigue Evaluation

- Fatigue is a material failure mechanism that can occur during dynamic loading scenarios that involve a cycling, reversing load that continues over a length of time.
- Fatigue failure can occur when the loads are below their normal static failure strength.
- Typical fatigue evaluations assume that fatigue damage accumulates over time according to Miner's Rule.
 - Every strain cycle removes a fraction of fatigue life.
 - S-N curves relate strain amplitudes (S) to number of cycles to failure (N).
- Cladding fatigue evaluations were made using the strain gage data collected in the MMTT.
 - Accounting for real irradiated rods is done by adjusting the fatigue histogram.

Fatigue Curves for Irradiated Zircaloy



<u>O'Donnell</u>: S-N curve used to define accumulated damage in this work. O'Donnell WJ and BF Langer. 1964. "Fatigue design basis for zircaloy components." *Nuclear Science and Engineering* (20):1–12

<u>NRC</u>: S-N curve based on CIRFT data. Defined in NUREG-2224, Dry Storage and Transportation of High Burnup Spent Nuclear Fuel. Draft for Comment, 2018.

Cladding Fatigue Summary



Westbound rail total accumulated fatigue damage for each strain gage. All damage fractions are less than 1E-10, when a damage fraction of 1.0 indicates fatigue failure. Damage is calculated according to Miner's Rule, ASTM rainflow counting, and the O'Donnell irradiated zircaloy S-N curve.

Example Fatigue Cycle Histogram

Full westbound rail strain gage strain cycle counts.



All strain cycles are rounded up to nearest integer for histogram binning.

How to Account for Irradiated Fuel?

- Test as an Analog
 - Assume the as-tested fuel rods are a perfect analog for irradiated fuel.
- Adjust the Test Data to Account for Differences
 - Stress concentration in cladding between pellets.
 - Change in fuel rod stiffness changes natural frequency.
 - Increase number of cycles.
 - Increase vibration amplitudes.
 - Modeling basis.
 - Transmissibility basis.
- Recognize that adjusting vibration amplitudes too high can lead to impractical fuel rod deflections.
 - Space inside fuel basket is limited, so deflections are limited.

Adjusting Accumulated Fatigue Damage for Irradiated Fuel

| | Total Strain Amplitude Adjustment Factor | Cycle Increase Factor | Adjusted Damage Fraction | Notes |
|--------------------------------------|---|-----------------------------|--------------------------------|--|
| Test as Analog | 1.0 | 1.0 | 4.5E-11 | Does not account for irradiated condition. |
| Model-Based Estimate | 2.1 | 1.5 | 5.5E-9 | Modeling-based adjustments. No practical difference from analog case. |
| Bounding Estimate Irradiated Fuel | 14 | 1.5 | 4.1E-4 | Bounding because of fuel rod deflection limits. |
| Resonance 3% Damping | 23.8 | 1.5 | 9.5E-3 | Not practical. Fuel rod deflection exceeds available space. |
| Resonance 1% Damping | 70 | 1.5 | 1.6 | Not Practical. Fuel rod deflection exceeds available space. |

- There is no practical difference in fatigue damage between the test as an analog or adjusting the test data to represent the irradiated condition.
 - The vibration energy is so low, the potential amplification does not make a difference.
- The bounding estimate assumes a higher amplification based on an alignment of natural frequencies.
 - This case would require about 5mm of fuel rod deflection. Available space is less than this.
- The resonance cases are only listed to show how much amplification is needed to challenge the fatigue limits.
 - The 1% damping case needs about 25mm of room to deflect.

Next Steps – FY19 (1 of 3)

- Atlas railcar analysis
 - Apply the modeling experience of the test campaign to the Atlas railcar.
 - Check the dynamic behavior of the many cask and cradle configurations using structural-dynamic modeling.
 - 4 cradle designs, many cask designs with different masses, all with potentially different dynamic responses
 - Confirm that cladding strains are low in these configurations



Family 1AREVA-TN: TN-40, TN-40HT, TN-32B
Holtec: Hi-Star 60, Hi-Star 100, Hi-Star 100HB, Hi-Star 180Family 2AREVA-TN: TN-68
NAC: MAGNATRAN, NAC-STC, NAC-UMSFamily 3AREVA-TN: MP197, MP197HB
Energy Solutions: TS125Family 4AREVA-TN: MP197HB

Atlas Rail Car Cradle Configurations

Next Steps – FY19 (2 of 3)

Evaluate the Effect of Rubber Pads in the Test Configuration.



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Next Steps – FY19 (3/3)

- Canister system analysis
 - The test used a bare fuel system.
 - Check the dynamic behavior of a fuel canister system with structural-dynamic modeling.
 - Confirm that cladding strains are low in these configurations
- Alternate fuel assembly analysis
 - The test used 17x17 PWR fuel.
 - Check BWR fuel and other PWR fuel with structural-dynamic modeling.
 - Confirm that cladding strains are low in these configurations
- Evaluate information coming from sister rod tests
 - Check potential stress concentrations caused by large pellet gaps with structural-dynamic modeling.

Conclusions

- The shock and vibration loads on the fuel rods recorded in the MMTT are approximately zero.
 - The cladding strains are below normal engineering notice.
 - Cladding fatigue damage is below the practical endurance limit.
 - The peak strain energy in the cladding is comparable to the kinetic energy of a raindrop.
- Structural-dynamic models are validated against test data.
 - Ready to evaluate other conveyance systems and fuel types.
- Ongoing work will evaluate the effect of rubber pads on the test data.
- Ongoing work expected to confirm that changes in conveyance system design or fuel assembly design will not affect these conclusions.

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Normal Conditions of Transport (NCT) Shock and Vibration Loads

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Irradiated fuel rod conditions and structural capacity >> NCT transportation loads, therefore it is safe to ship UNF and to expect it to arrive with no additional damage.

Questions?

Clean. Reliable. Nuclear.

Backup Slides

 These slides are included for potential reference during Q&A.

Rainflow Counting

- Is a method to determine the number of fatigue cycles and amplitudes in a specified time interval for variable loadings
- For simple periodic loadings rain-flow counting is unnecessary
- Developed and validated MATLAB code for Rain-flow counting algorithm last summer
- Simple steps listed in website of American Society of Testing and Materials (ASTM E1049)



Fatigue Damage Calculation



Where

A & B = constants

 $S_i = strain amplitude$

 $n = Number of cycles accumulated at stress S_i$

N = Average number of cycles till failure at the ith stress S_i

Example Fatigue Cycle Histogram

Full westbound rail strain gage strain cycle counts.



All strain cycles are rounded up to nearest integer for histogram binning.

because strain cycles of this amplitude could not possibly contribute a 0.01 damage fraction in a practical cross country trip.

line noise.

How Many Relevant Strain Cycles?

| 3,800 Cycles > 10uE | | # of Strain Cycles (Failure) | Strain Amplitude (uE) | Cycles to Reach a Damage Fraction of 0.01 | |
|----------------------------|---|------------------------------------|-----------------------------|---|----------------------------|
| in Westbound Rail | | 10,000 | 1,160 | 100 | |
| | | 100,000 | 786 | 1,000 | |
| | | 1.0E+06 | 533 | 10,000 | If 1E+8 cycles were 112 uE |
| 1E+8 Total Cycles in | | 1.0E+07 | 361 | 100,000 | or higher, the accumulated |
| Westbound Rail | | 1.0E+08 | 245 | 1.0E+06 | damage could be affected |
| (Any amplitude) | | 1.0E+09 | 166 | 1.0E+07 | by 0.01 or higher. |
| | | 1.0E+10 | 112 | 1.0E+08 | |
| | | 1.0E+11 | 76 | | |
| | | 1.0E+12 | 52 | 1.0E+10 | |
| | | 1.0E+13 | 35 | 1.0E+11 | |
| | | 1.0E+14 | 24 | 1.0E+12 | |
| | | 1.0E+15 | 16 | 1.0E+13 | |
| | _ | 1.0E+16 | 11 | 1.0E+14 | |
| | | 1.0E+17 | 7 | 1.0E+15 | Why count any cycle |
| | | 1.0E+18 | 5 | 1.0E+16 | below 10 uE2 |
| | | 1.0E+19 | 3 | 1.0E+17 | |
| | | 1.0E+20 | 2 | 1.0E+18 | There are not enough |
| | | 1.0E+21 | 1.6 | 1.0E+19 | domage fraction by any |
| Note: O'Donnell Irradiated | | 1.0E+22 | 1.1 | 1.0E+20 | practical amount |
| Zircaloy S-N Design Curve | : | 1.0E+23 | 0.7 | 1.0E+21 | |

Adjusting Accumulated Fatigue Damage for Irradiated Fuel

Adjust As-Tested Rods to Real Irradiated Fuel Rods

| | Stress Conc. Factor | El Strain Amplitude Adjust. Factor | Total Strain Amplitude Adjust. Factor | Cycle Increase Factor | Adjusted Damage Fraction |
|--|---------------------------|---|---|-----------------------------|--------------------------------|
| Test as Analog | 1.0 | 1.0 | 1.0 | 1.0 | 4.5E-11 |
| Model Estimate | 1.4 [1] | 1.5 [2] | 2.1 | 1.5 [3] | 5.5E-9 |
| Bounding Estimate Irradiated Fuel | 1.4 | 10 [4] | 14 | 1.5 | 4.1E-4 |
| Resonance 3% Damping | 1.4 | 17 [4] | 23.8 | 1.5 | 9.5E-3 |
| Resonance 1% Damping | 1.4 | 50 [4] | 70 | 1.5 | 1.6 |

[1] Adkins H, K Geelhood, B Koeppel, J Coleman, J Bignell, G Flores, J-A Wang, S Sanborn, R Spears, and N Klymyshyn. 2013. Used Nuclear Fuel Loading and Structural Performance Under Normal Conditions of Transport – Demonstration of Approach and Results on Used Fuel Performance Characterization. FCRD-UFD-2013-000325, Pacific Northwest National Laboratory, Richland, Washington.

[2] Klymyshyn NA, PJ Jensen, and NP Barrett. 2015. Shaker Table Modeling Support Task 2015. PNNL-24735. Pacific Northwest National Laboratory, Richland, Washington.

[3] Klymyshyn NA, PJ Jensen, and NP
Barrett. 2016b "Modeling Used Fuel
Response to 30 cm Package Drops."
Presented by Ross at 18th International
Symposium on Packaging and Transportation
of Radioactive Materials PARTRAM 2016,
Kobe Japan, Japan. PNNL-SA-120852.

[4] Transmissibility at resonance: see next slide

Transmissibility at Resonance

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10

0.1

SHOCK AND VIBRATION RESPONSE SPECTRA COURSE

Unit 8. Transmissibility Function for Acceleration

By Tom Irvine

www.vibrationdata.com

Closed-form solution of transmissibility magnitude:

$$\left| \begin{array}{c} \frac{\ddot{x}}{\ddot{y}} \\ \end{array} \right| = \sqrt{\frac{1 + (2\xi\rho)^2}{\left(1 - \rho^2\right)^2 + (2\xi\rho)^2}} \ , \label{eq:constraint}$$

 $\rho = f / f_n$

0.5

TRANSMISSIBILTY MAGNITUDE SDOF SYSTEM SUBJECTED TO BASE EXCITATION

FREQUENCY RATIO (f / fn)

1

2

10

Q=1 Q=2 Q=10

2ξ

Stress Concentrations at Pellet Gaps (1.4x)

We will assume that the strain gages were not placed at anticipated stress concentrations. FCRD-UFD-2013-000325 predicted these stress concentrations could be on the order of 1.4x the nominal strain. Fatigue amplitudes will be increased by this factor. (Image from FCRD-UFD-2013-000325)



Data Analysis Study: Strain Gage Response Correlation, by Frequency

Average signal correlation when low pass frequency filter applied to strain gage data.

| | 15 Hz | 30 Hz | 45 Hz | 60 Hz | 75 Hz | 90 Hz | Baseline (200 Hz) |
|---|-------|-------|-------|-------|-------|-------|----------------------|
| Top Mid- Span Strain Gage, All Three Assemblies | 0.98 | 0.93 | 0.76 | 0.60 | 0.54 | 0.51 | 0.50 |

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Signal correlation: 3 strain gages in similar locations on all 3 fuel assemblies. 10 second open rail event.





Initial Model Architecture – 3 Parts



50 Mph Pitch and Bounce: Test Data Summary

Maximum Recorded Strains (uE) Per Rod 50 mph Pitch and Bounce Test

| | Max | LC | М | RC |
|--------|-----|----|----|----|
| SNL | 19 | 13 | 18 | 19 |
| ENSA | 10 | 10 | 10 | 10 |
| Korean | 5 | - | 5 | - |

Irradiated Cladding Yield ~10,000 uE

Strain Data Summary:

- P&B test data is 5 to 19 uE
- SNL assembly is 13 to 19 uE

LC Μ RC Guide Tubes Instrumented Rods

Instrumented Fuel Rod Map

Model Results: Cladding Strain Pretest Prediction (50 mph P&B)

Model Calculated Peak Strain (uE) Per Rod

| 102 | 104 | 95 | 101 | 95 | 80 | 87 | 98 | 112 | 91 | 107 | 95 | 85 | 83 | 89 | 102 | 88 |
|-----|-----|-----|-----|-----|-----|-----|-----|-----|-----|-----|-----|-----|-----|-----|-----|-----|
| 89 | 92 | 88 | 89 | 99 | 95 | 115 | 101 | 108 | 102 | 91 | 115 | 102 | 79 | 86 | 93 | 92 |
| 91 | 100 | 111 | 119 | 93 | | 116 | 92 | | 106 | 106 | | 97 | 102 | 106 | 85 | 92 |
| 98 | 85 | 83 | | 106 | 221 | 147 | 100 | 206 | 126 | 93 | 196 | 131 | | 99 | 83 | 87 |
| 89 | 92 | 86 | 106 | 113 | 99 | 79 | 83 | 112 | 102 | 91 | 118 | 119 | 115 | 89 | 102 | 86 |
| 90 | 90 | | 98 | 91 | | 83 | 87 | | 102 | 89 | | 109 | 88 | | 115 | 82 |
| 85 | 85 | 276 | 126 | 118 | 195 | 110 | 88 | 264 | 116 | 89 | 212 | 131 | 85 | 268 | 116 | 87 |
| 89 | 92 | 107 | 82 | 83 | 100 | 87 | 84 | 86 | 95 | 86 | 101 | 91 | 92 | 93 | 84 | 77 |
| 81 | 96 | | 98 | 85 | | 97 | 88 | | 109 | 86 | | 98 | 85 | | 93 | 88 |
| 85 | 90 | 336 | 156 | 88 | 238 | 124 | 101 | 193 | 130 | 92 | 238 | 131 | 99 | 225 | 122 | 95 |
| 93 | 95 | 99 | 129 | 87 | 102 | 86 | 93 | 103 | 86 | 91 | 125 | 86 | 88 | 104 | 100 | 89 |
| 89 | 96 | | 93 | 97 | | 113 | 84 | | 91 | 83 | | 99 | 102 | | 96 | 92 |
| 86 | 122 | 150 | 89 | 111 | 318 | 140 | 86 | 172 | 117 | 105 | 226 | 109 | 92 | 97 | 109 | 97 |
| 89 | 82 | 97 | | 89 | 107 | 98 | 91 | 112 | 94 | 94 | 109 | 108 | | 103 | 87 | 86 |
| 111 | 91 | 110 | 174 | 100 | | 95 | 108 | | 107 | 95 | | 104 | 208 | 111 | 93 | 131 |
| 89 | 116 | 71 | 113 | 109 | 179 | 123 | 104 | 194 | 112 | 119 | 214 | 132 | 89 | 88 | 92 | 112 |
| 113 | 96 | 102 | 114 | 83 | 80 | 113 | 100 | 94 | 99 | 96 | 83 | 85 | 98 | 87 | 89 | 89 |
| | | | | | | | | | | | | | | | | |

Results Documented:

• SFWD-SFWST-2017-000035

Model Prediction:

- Most rods about 100 uE
- Max rod is 336 uE (outlier)

Compare to Test Data:

- Test data is 5 to 19 uE
- Add gravity strain (~50 uE)

Review Conclusion:

Not working as intended

Model Fixes Needed:

- Double Precision Solver
- Damping/Numerical Noise
- Initialization/Gravity

Instrumented Rods

Maximum Calculated Strain Rod

Guide Tubes

Model Results: Revised Post-Test Model (50 mph P&B)

Post Test Model Validation:

- Resolved model flaws and adjusted results for gravity
- Peak strain: 33 uE (calculated)
- Instrumented rod strains: 12-14 uE (calculated)
- Measured Strains: 5-19 uE (measured)
- Good model, but not necessary when strains are this low.

Maximum Recorded Strains (uE) Per Rod 50 mph Pitch and Bounce Test

| | Max | LC | М | RC |
|--------|-----|----|----|----|
| SNL | 19 | 13 | 18 | 19 |
| ENSA | 10 | 10 | 10 | 10 |
| Korean | 5 | - | 5 | - |

| 12 | 12 | 12 | 12 | 12 | 12 | 12 | 12 | 12 | 12 | 12 | 12 | 12 | 12 | 12 | 12 | 12 |
|----|----|----|----|----|----|----|----|----|----|----|----|----|----|----|----|----|
| 12 | 12 | 12 | 12 | 6 | 17 | 10 | 6 | 14 | 8 | 6 | 14 | 8 | 12 | 12 | 12 | 12 |
| 12 | 12 | 7 | 19 | 8 | | 11 | 7 | | 8 | 9 | | 8 | 17 | 12 | 12 | 12 |
| 12 | 12 | 12 | | 14 | 17 | 25 | 17 | 15 | 23 | 23 | 17 | 8 | | 10 | 11 | 11 |
| 11 | 6 | 5 | 6 | 14 | 19 | 7 | 6 | 19 | 9 | 5 | 16 | 20 | 5 | 7 | 8 | 11 |
| 11 | 5 | | 12 | 7 | | 9 | 7 | | 8 | 8 | | 10 | 5 | | 7 | 11 |
| 11 | 15 | 13 | 19 | 16 | 12 | 23 | 17 | 13 | 20 | 16 | 15 | 23 | 16 | 11 | 18 | 11 |
| 10 | 6 | 13 | 8 | 7 | 20 | 8 | 7 | 17 | 8 | 6 | 14 | 8 | 6 | 14 | 9 | 11 |
| 10 | 6 | | 9 | 6 | | 10 | 7 | | 9 | 8 | | 9 | 8 | | 8 | 10 |
| 10 | 16 | 8 | 21 | 16 | 9 | 22 | 17 | 12 | 20 | 17 | 9 | 19 | 16 | 7 | 20 | 10 |
| 10 | 6 | 18 | 9 | 6 | 15 | 8 | 6 | 18 | 8 | 6 | 16 | 8 | 6 | 16 | 8 | 10 |
| 10 | 5 | | 10 | 11 | | 12 | 7 | | 12 | 10 | | 9 | 7 | | 8 | 10 |
| 11 | 14 | 6 | 7 | 13 | 8 | 20 | 18 | 11 | 22 | 17 | 8 | 12 | 7 | 6 | 17 | 11 |
| 11 | 11 | 7 | | 7 | 20 | 15 | 6 | 18 | 12 | 11 | 18 | 6 | | 9 | 11 | 11 |
| 11 | 11 | 15 | 11 | 9 | | 12 | 8 | | 11 | 10 | | 11 | 7 | 18 | 11 | 11 |
| 11 | 11 | 10 | 11 | 14 | 11 | 33 | 18 | 11 | 22 | 17 | 8 | 24 | 11 | 10 | 11 | 11 |
| 11 | 11 | 11 | 11 | 13 | 13 | 12 | 12 | 12 | 13 | 13 | 11 | 11 | 11 | 10 | 11 | 11 |

Four Atlas Railcar Cradles



NWTRB Questions

1:10 p.m. ENSA Cask Multimodal Transportation Test and Related Structural Modeling and Analysis

Sylvia Saltzstein, Sandia National Laboratories, and *Nicholas Klymyshyn*, Pacific Northwest National Laboratory

- a. In a summary fashion, what is the progress on the Equipos Nucleares, S.A. (ENSA) cask transportation test?
- b. How were the effects of using surrogate components in the ENSA cask test evaluated to determine if the test results can be applied to a real transportation system? How does the use of a pad, which was placed between the cradle and cask in the test, but which may not be present or may be of different material in other transportation operations, affect the applicability of the ENSA test results to different transportation systems?
- c. How will the behavior of real irradiated spent nuclear fuel during transport be evaluated?
- d. Considering the relatively large transient impulse loads measured during the rail testing at the Transportation Technology Center compared to the actual rail vibration loads, explain how the effects of large transient impulses will be accounted for when determining the fatigue lifetime of spent nuclear fuel.
- e. Was the frequency spectrum observed for the surrogate fuel rods in the ENSA cask test consistent with the frequency spectrum used for the fatigue testing at Oak Ridge National Laboratory? If not, what are the implications of the inconsistency for using the fatigue testing results to model actual transportation operations?

1:50 p.m. Questions, discussion